### Department of Mechanical and Aerospace Engineering 2008 Ph.D Qualifying Exam Jan 28, 2008

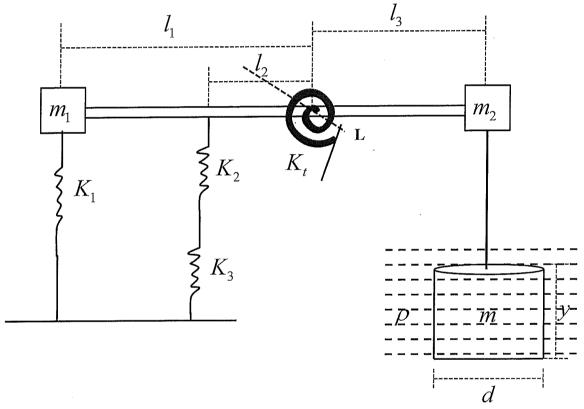
# **Dynamics and Control**

Assigned	#	

# NO TEXTBOOKS OR NOTES ALLOWED. YOU ARE REQUIRED TO ANSWER 4 OF THE 6 QUESTIONS.

#### Problem A:

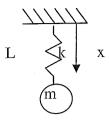
Derive the equation of motion of the system shown in the figure below.



What is the damping ratio and the natural frequency of the system?

#### **Problem B:**

Consider the spring mass system shown below. The spring has a mass density of  $\delta$  kg/m in its un-deformed state, and an un-deformed length of L. Derive the equations of motion and determine the natural frequency of the system.



## **Problem C:**

Consider the following partitioned system matrix:

$$A = \begin{bmatrix} A_{11}(t) & A_{12}(t) \\ 0 & A_{22}(t) \end{bmatrix}$$

where  $A_{11}(t)$ ,  $A_{22}(t)$  and  $A_{12}(t)$  are generally time-varying sub-matrices (they cannot be assumed to be scalars).

a) Prove that the state transition matrix has the form

$$\Phi(t, t_0) = \begin{bmatrix} \Phi_{11}(t, t_0) & \Phi_{12}(t, t_0) \\ 0 & \Phi_{22}(t, t_0) \end{bmatrix}$$

$$\begin{aligned} \text{where } \left( \partial / \partial t \right) \Phi_{ii} \left( t, t_0 \right) &= A_{ii} \left( t \right) \Phi_{ii} \left( t, t_0 \right), i = 1, 2 \text{ and} \\ \left( \partial / \partial t \right) \Phi_{12} \left( t, t_0 \right) &= A_{11} \left( t \right) \Phi_{12} \left( t, t_0 \right) + A_{12} \left( t \right) \Phi_{22} \left( t, t_0 \right), \text{ with } \Phi_{12} \left( t_0, t_0 \right) &= 0. \end{aligned}$$

b) Use this result to find  $\Phi(t,0)$  for

$$A = \begin{bmatrix} -1 & e^{2t} \\ 0 & -1 \end{bmatrix}$$

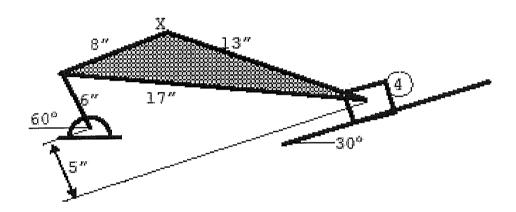
Prove that your answer is correct.

#### Problem D:

Consider the four-link mechanism shown in the figure below. Determine the maximum displacement of link 4.

The mechanism is driven by a D.C. motor attached to the crank with an angular velocity of 60 rpm clockwise.

Determine the velocity of the link 4 when the crank rotates at 60 rpm for the link locations shown below.

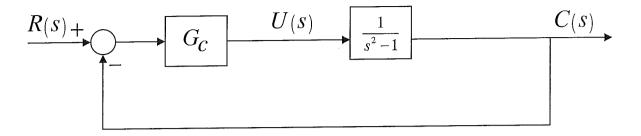


#### **Problem E:**

Consider the following open-loop system:

$$\frac{C(s)}{U(s)} = \frac{1}{s^2 - 1}$$

- With an impulse input for u(t) and zero initial conditions, solve for c(t).
- Draw the Bode plot for the open-loop system.
- Suppose we multiply the open-loop transfer function by a constant. What does this do to the magnitude and phase of the Bode plot?
- Can the system be controlled from an asymptotic sense using proportional control only? Explain this answer using 1) a phase margin argument, as well as 2) from the closed-loop tansfer.
- Design a compensator  $G_c$  to stabilize the system so that the closed-loop system has a natural frequency of 2 rad/sec and a damping ratio of  $\sqrt{2}/2$ .



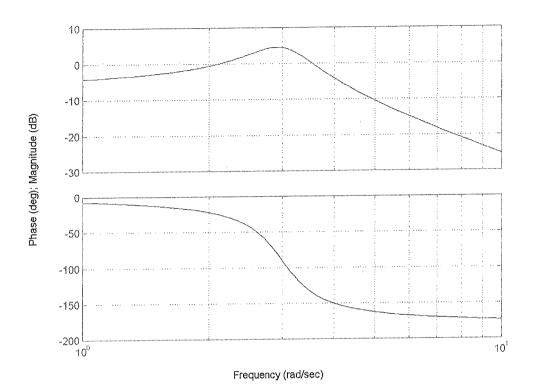
#### Problem F:

A linear, time-invariant, SISO (single input - single output) system is represented by the Bode plots shown below. Answer the following questions:

- a) Estimate the system's transfer function
- b) Is this system stable, marginally stable, or unstable? Explain.
- c) What is the system's output if the input is  $u = 5 * \sin (4t)$ ?
- d) If the input is  $u = 10 * \sin(w*t)$ , for what value(s) of w (if any) is the system output magnitude exactly equal to 10?
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- j) What is the system's approximate natural frequency (if any)?

# What is the system's approximate damping ratio?

#### Bode Diagrams



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# **EXAMINATION**

MECHANICAL AND AEROSPACE ENGINEERING
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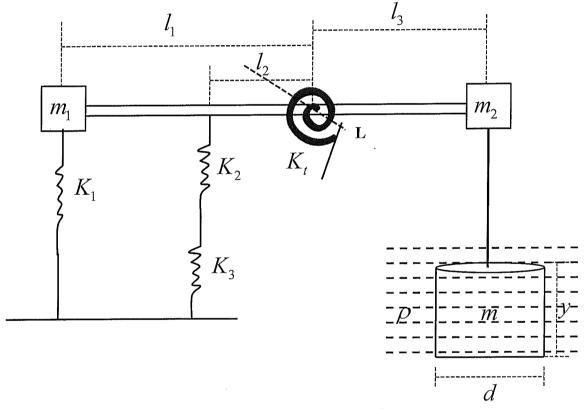
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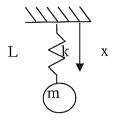
Derive the equation of motion of the system shown in the figure below.



What is the damping ratio and the natural frequency of the system?

#### Problem B:

Consider the spring mass system shown below. The spring has a mass density of  $\delta$  kg/m in its un-deformed state, and an un-deformed length of L. Derive the equations of motion and determine the natural frequency of the system.



## **Problem C:**

Consider the following partitioned system matrix:

$$A = \begin{bmatrix} A_{11}(t) & A_{12}(t) \\ 0 & A_{22}(t) \end{bmatrix}$$

where  $A_{11}(t)$ ,  $A_{22}(t)$  and  $A_{12}(t)$  are generally time-varying sub-matrices (they cannot be assumed to be scalars).

a) Prove that the state transition matrix has the form

$$\Phi(t,t_{0}) = \begin{bmatrix} \Phi_{11}(t,t_{0}) & \Phi_{12}(t,t_{0}) \\ 0 & \Phi_{22}(t,t_{0}) \end{bmatrix}$$

$$\begin{split} \text{where } \left(\partial/\partial t\right)\Phi_{ii}\left(t,t_{0}\right)&=A_{ii}\left(t\right)\Phi_{ii}\left(t,t_{0}\right),\,i=1,2 \text{ and } \\ \left(\partial/\partial t\right)\Phi_{12}\left(t,t_{0}\right)&=A_{11}\left(t\right)\Phi_{12}\left(t,t_{0}\right)+A_{12}\left(t\right)\Phi_{22}\left(t,t_{0}\right),\,\text{with } \Phi_{12}\left(t_{0},t_{0}\right)&=0 \;. \end{split}$$

b) Use this result to find  $\Phi(t,0)$  for

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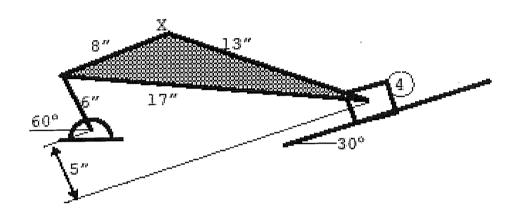
Prove that your answer is correct.

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Consider the four-link mechanism shown in the figure below. Determine the maximum displacement of link 4.

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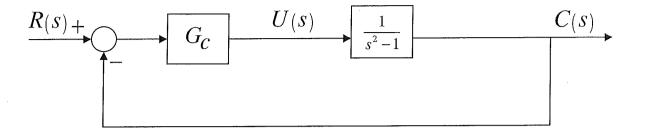


## Problem E:

Consider the following open-loop system:

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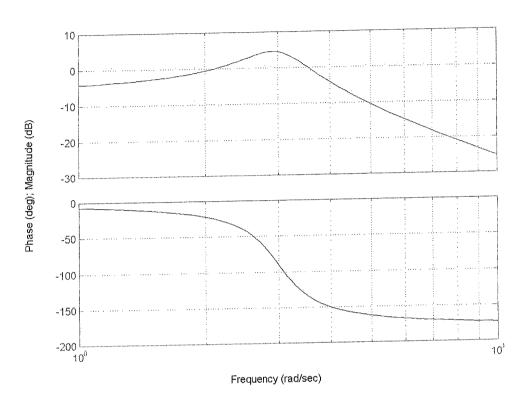
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#### Bode Diagrams



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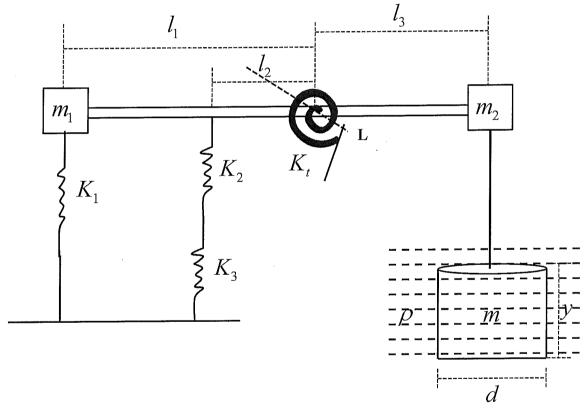
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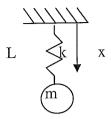
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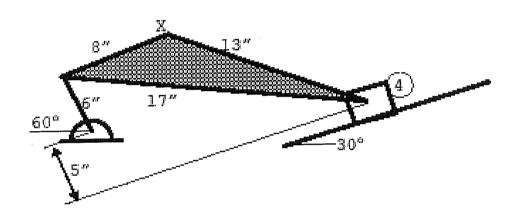
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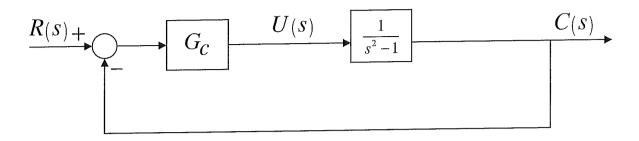


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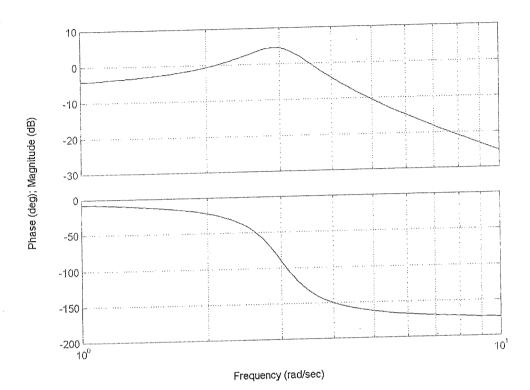
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## Bode Diagrams



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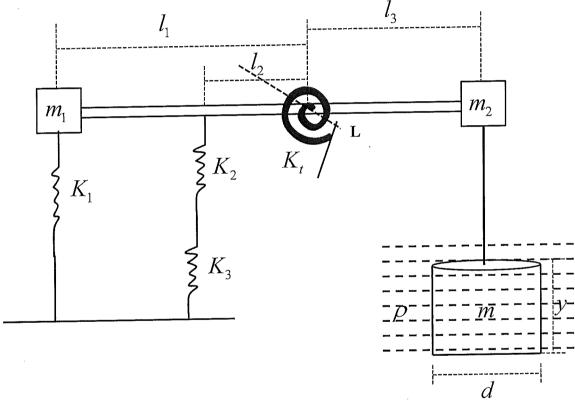
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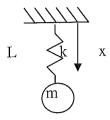
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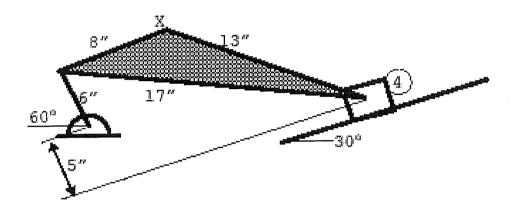
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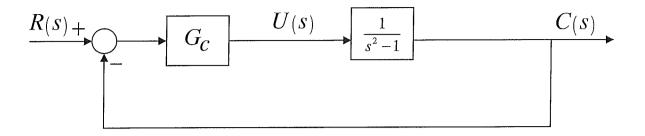


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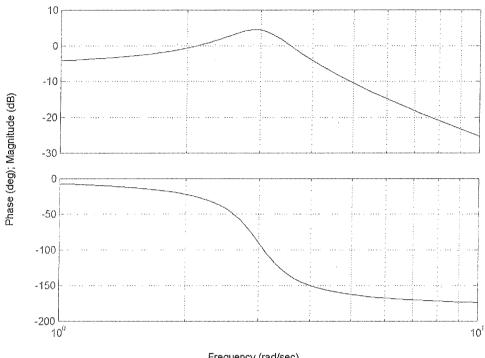
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#### Bode Diagrams



Frequency (rad/sec)